



Sanjay Ghodawat University, Kolhapur

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2018-19
EXM/P/09/01

Year and Program: 2018-19

School of Technology

Department of Mechanical Engineering

Course Code: MET209

Course Title: Strength of Materials

Semester – III

Day and Date

Friday
14-06-2019

End Semester Examination (ESE)

Time: 3 Hrs. Max Marks: 100
2.30 to 5.30 PM.

Instructions:

- 1) All questions are compulsory.
- 2) Assume suitable data wherever necessary.
- 3) Figures to the right indicate full marks.

Q.1

a) Define

- i. Volumetric Strain
- ii. Young's Modulus
- iii. Poisson's Ratio
- iv. Factor of Safety
- v. Modulus of Rigidity
- vi. Bulk Modulus

| Marks | Bloom's Level | CO |
|-------|---------------|-----|
| 06 | L2 | CO1 |

OR

A member ABCD is subjected to point loads P_1 , P_2 , P_3 and P_4 as shown in fig 1a. Calculate the force P_3 necessary for equilibrium if $P_1 = 120\text{kN}$, $P_2 = 22\text{kN}$ and $P_4 = 160\text{kN}$. Determine also the net change in length of the member. Take $E = 2 \times 10^5 \text{ N/mm}^2$

06 L3 CO1

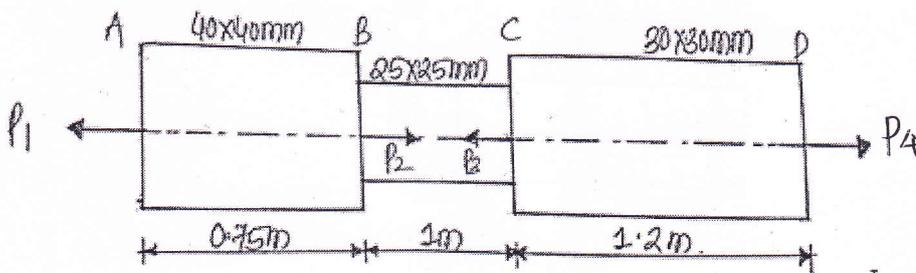


Fig.1a.

ESE

- b) Derive the expression for normal stress & tangential stress on any oblique section of a body subjected to direct stress in two mutually perpendicular directions. 09 L₃ CO2

OR

At a point within a body subjected to mutually perpendicular directions, the stresses are 80N/mm^2 tensile and 40N/mm^2 tensile. Each of the above stresses is accomplished by shear stress of 60N/mm^2 . Determine the normal stress, shear stress and resultant stress on an oblique plane inclined at an angle of 45° with the axis of minor tensile stress. 09 L₃ CO2

- Q.2 Draw the shear force & bending moment diagram for the beam as shown in fig. 2a. Also find the point of contraflexure if any, 16 L₃ CO3

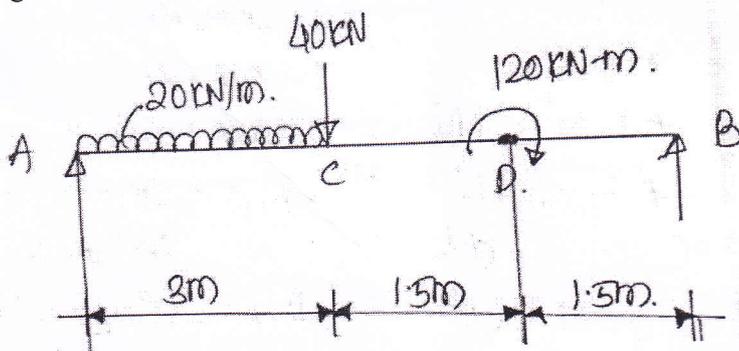


fig. 2a

- Q.3 If the shear force acting on the I section beam as shown in fig. 3 is 40kN , find the maximum shear stress developed in the I section. Also calculate the Shear Stresses at Neutral axis & also draw the shear stress distribution over the depth of the section. 15 L₃ CO4

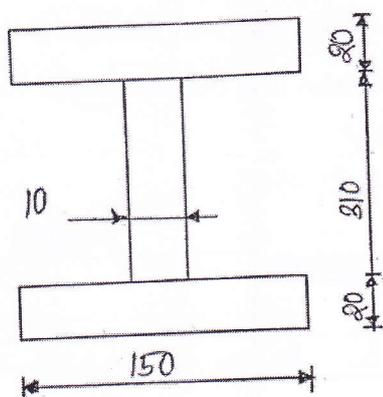


Fig 3

All Dimensions are in mm

Q.4 **Solve any Two**

- | | | | | |
|----|--|----|----------------|-----|
| a) | Derive the expression for slope & deflection of simply supported beam subjected to central point load. | 09 | L ₂ | CO5 |
| b) | A cantilever of length 'l' carries a uniformly distributed load of w/unit length over the span of the beam. Derive the relations for the slope of deflections for at free end. Use double integration method | 09 | L ₂ | CO5 |
| c) | Find the deflection at C in the beam loaded as shown in fig 4c, Take EI= 10,000kN-m ² | 09 | L ₂ | CO5 |

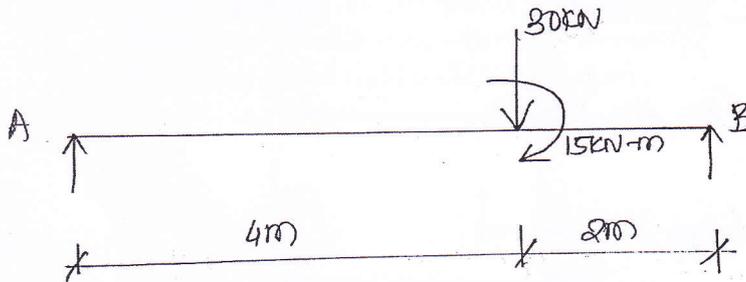


fig 4c.

Q.5 **Solve any Two**

- | | | | | |
|----|---|----|----------------|-----|
| a) | Derive the expression for the Euler's crippling load for long column with both the ends are hinged. | 09 | L ₃ | CO6 |
| b) | The cross section of the column is as shown in fig 5b. Calculate the length of the member for which crippling load by Euler's formula and Rankine formula will be same. Assume yield stress to be 330Mpa, E=210GPa, I _{xx} = 24.2x10 ⁶ mm ⁴ , I _{yy} = 3x10 ⁶ mm ⁴ & Rankines constant a as | 09 | L ₃ | CO6 |

$$a = \frac{1}{7500}$$

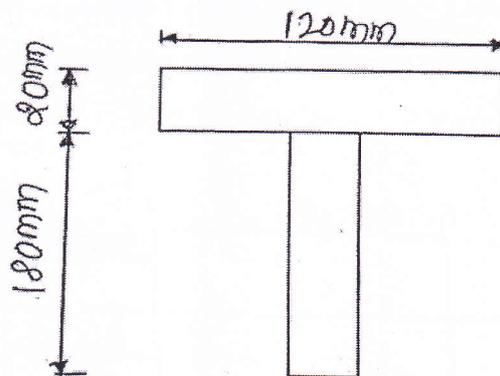


fig 5b

- c) A hollow circular shaft of 6m length and inner and outer diameters of 75mm and 100mm is subjected to a torque of 10kN-m. If $G=80\text{Gpa}$, Determine the maximum shear stress produced and the total angle of twist. 09 L₃ CO6

Q.6 Solve any Three

- a) Determine the maximum deflection for the beam shown in fig 6a, by double integration method. Take $EI=36000\text{kN-m}^2$ 06 L₃ CO5

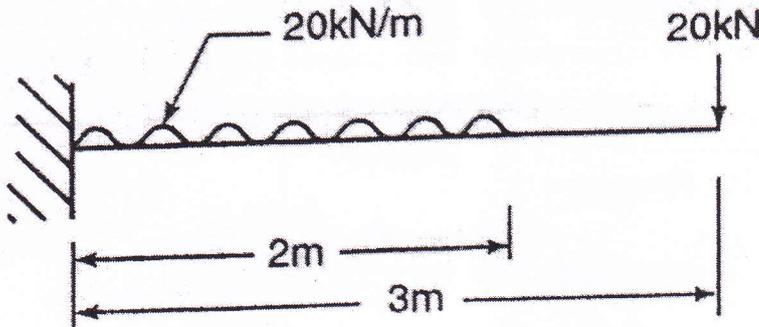


fig. 6a

- b) State the assumptions made in pure torsion 06 L₂ CO6
- c) Derive the torsional equation for the shaft L , radius R fixed at one end & subjected to torque T at the other end 06 L₂ CO6
- d) Explain the various end conditions and corresponding equivalent lengths of columns. 06 L₂ CO6
